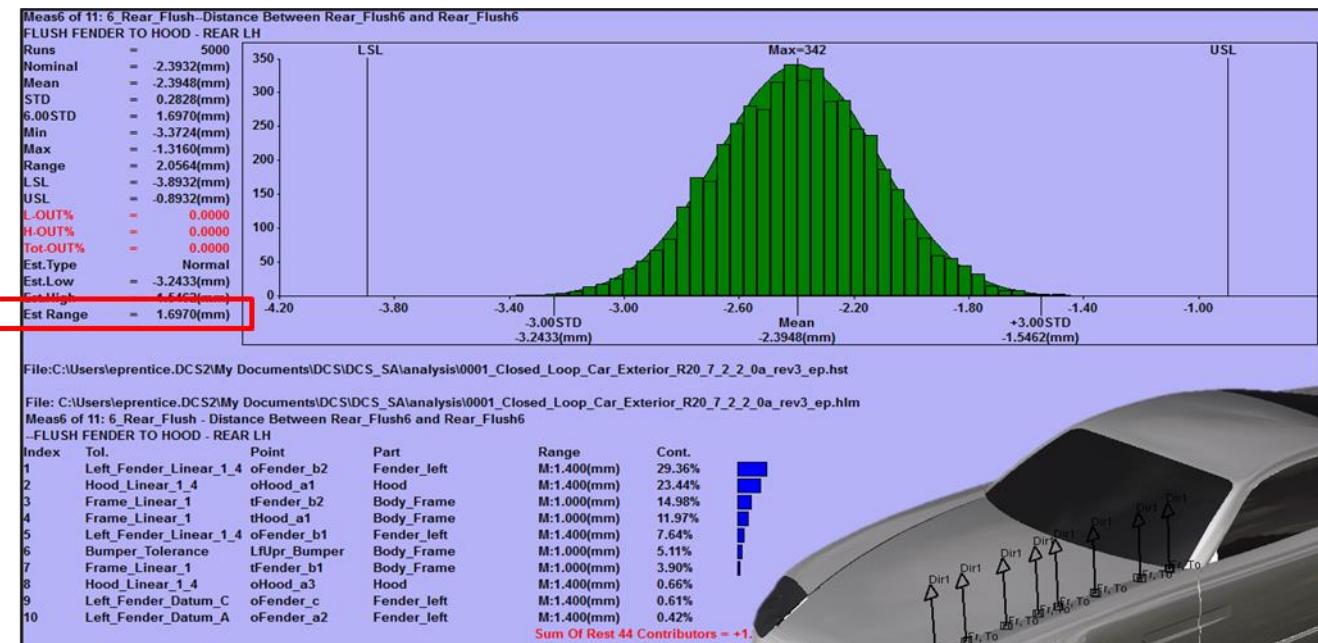
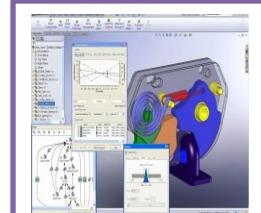


# Non-Rigid Tolerance Variation



# מי אנחנו? TES-RnD



## תחומי פעילות

- פיתוח השתלמויות בתחום הטולרנסים הגיאומטריים
- ייעוץ והדרבה בתחום הטולרנסים הגיאומטריים לחברות בארץ ו בחו"ל
- שירות הנדסה, תכנן ואנליזה (Tolerance Analysis)



## רקע וניסיון בטולרנסים גיאומטריים

- מוסמך ASME ברמת סニアר, 0716-S-DTP
- נציג ישראל ב TC213 ISO
- מעל עשר שנים ניסיון בהדרבה ויעוץ לחברות מובילות בארץ ו בחו"ל



## הסמכויות

- ISO 9001 - 2008
- ספק מוכבר משאבות"ט דרג A



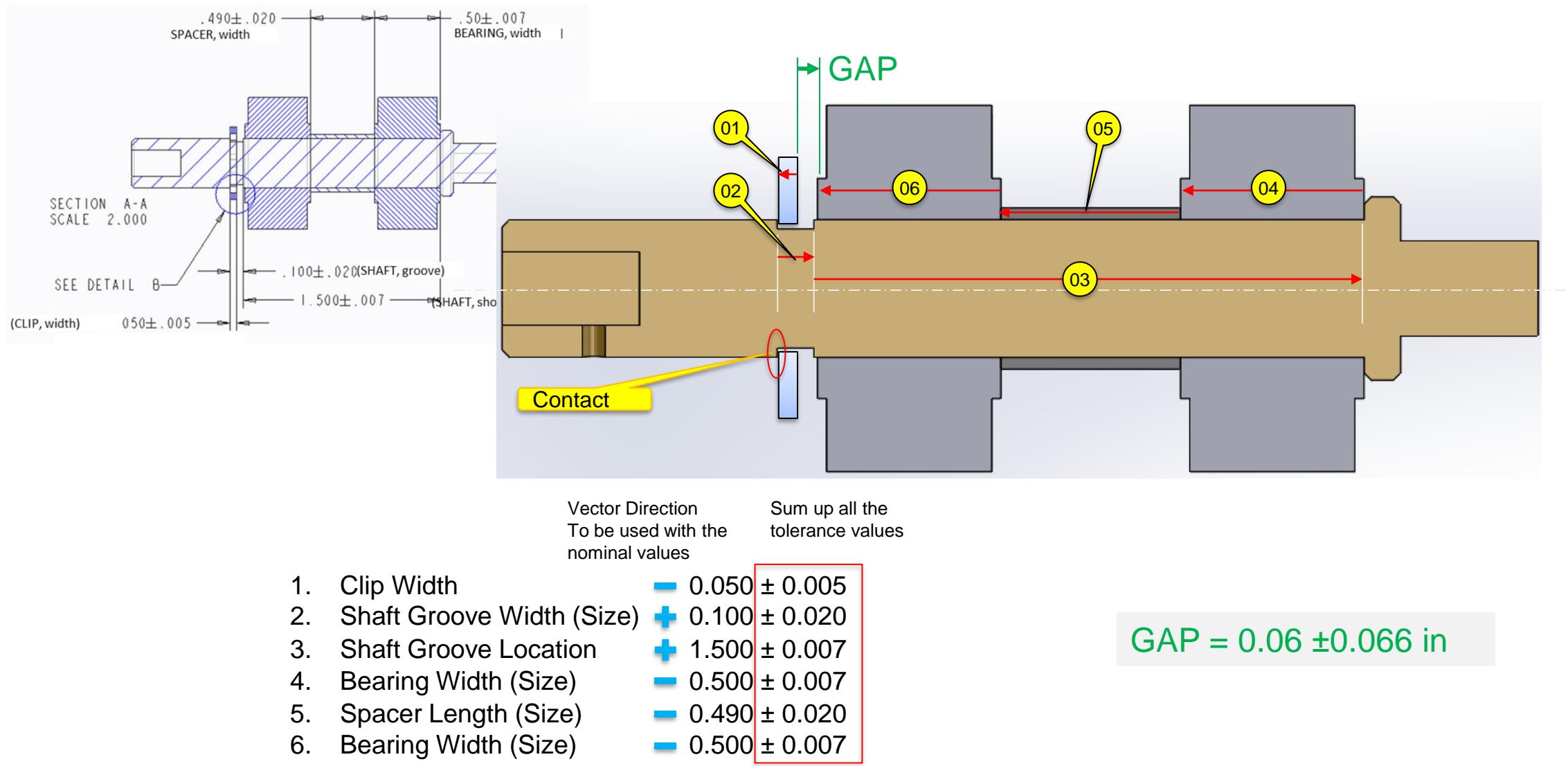
## יצוג שירות ותמיכה – תוכנה

- Tolerance Analysis 3DCS-DCS
- Tolerance Analysis CETOL-Sigmetrix
- - ניתוח תוצאות מדידה Smart Profile -Kotem

# Non-Rigid Tolerance Variation

- Tolerance Calculation & Simulation
- FEM – Rigid – Compliant
- Daimler Study
- Conclusions

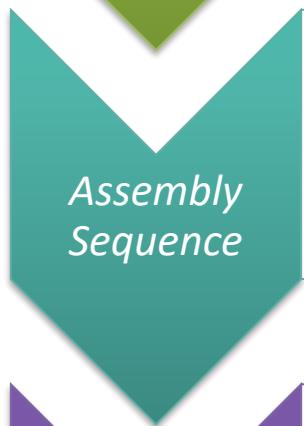
# Tolerance Stack



# Variation Simulation



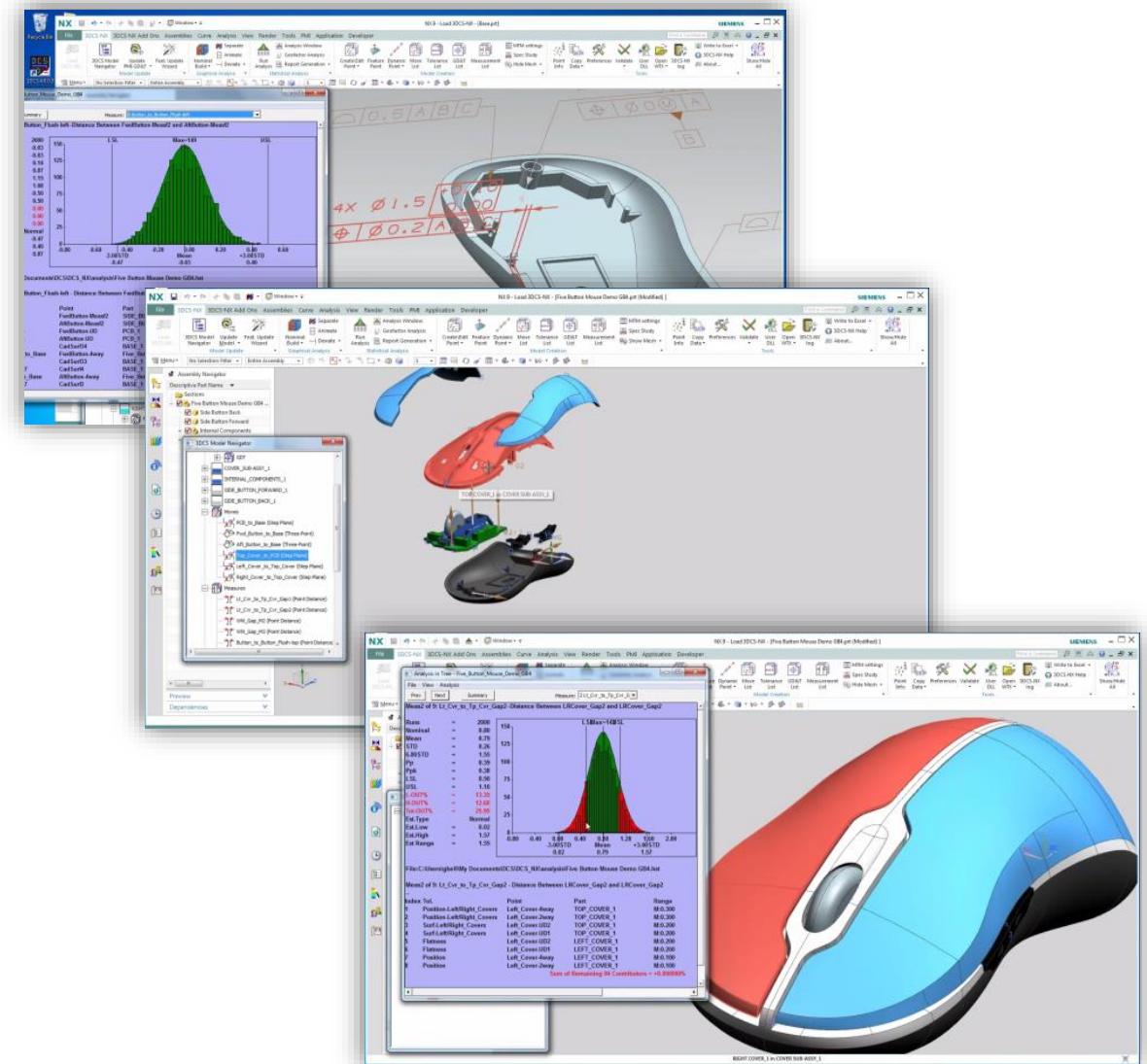
- Geometry
- GD&T
- Variation



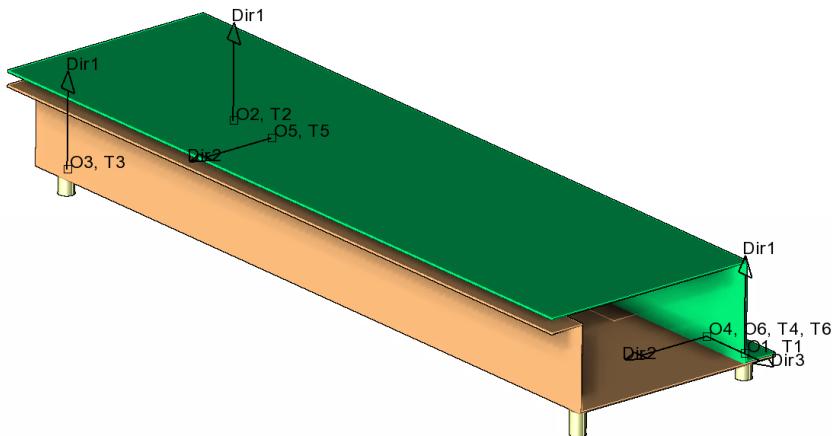
- Moves to assemble the parts



- Areas to analyze during simulation

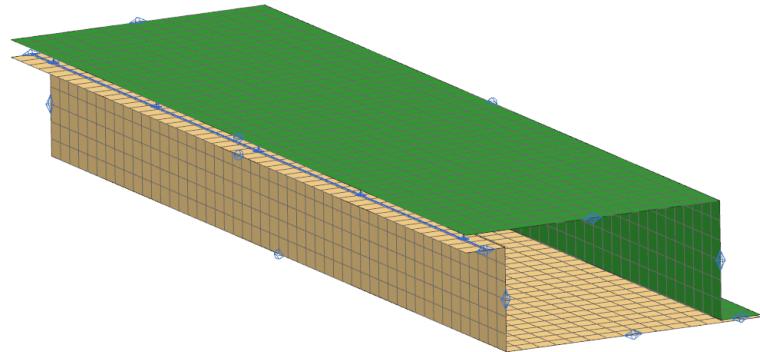


# Rigid & FEM Variation Simulation



## Rigid Tolerance Simulation

Simulates Tolerance Variation  
Ignores Elastic Constraints



## FEM Simulation

Simulates Elastic Constraints  
Ignores Tolerance Variation

Compliant Tolerance Simulation  
Simulates Elastic Constraints  
Simulates Tolerance Variation

# The Need

Simulate variation due to

- Process (welding, clamping, bolting and riveting)
- Environmental forces (Gravity, Heat)

Design to account for Variation

Optimize to reduce Process variation:

- Riveting sequences
- Clamps
- Bending.

# Daimler Study

ICED19 - International Conference on Engineering Design

5-8 AUGUST 2019

DELFT, THE NETHERLANDS



## A modeling approach for elastic tolerance simulation of the body in white hang-on parts

H. Zheng<sup>1</sup>, F. Litwa<sup>1</sup>, M. Bohn<sup>1</sup>, B. Reese<sup>2</sup>, C. Li<sup>2</sup>, K. Paetzold<sup>3</sup>

<sup>1</sup> Daimler AG, Sindelfingen, Germany

<sup>2</sup> Dimensional Control System Inc., Michigan, USA

<sup>3</sup> Bundeswehr University Munich, Neubiberg, Germany

Mercedes-Benz

The best or nothing.

ice<sup>19</sup>



Universität der Bundeswehr München

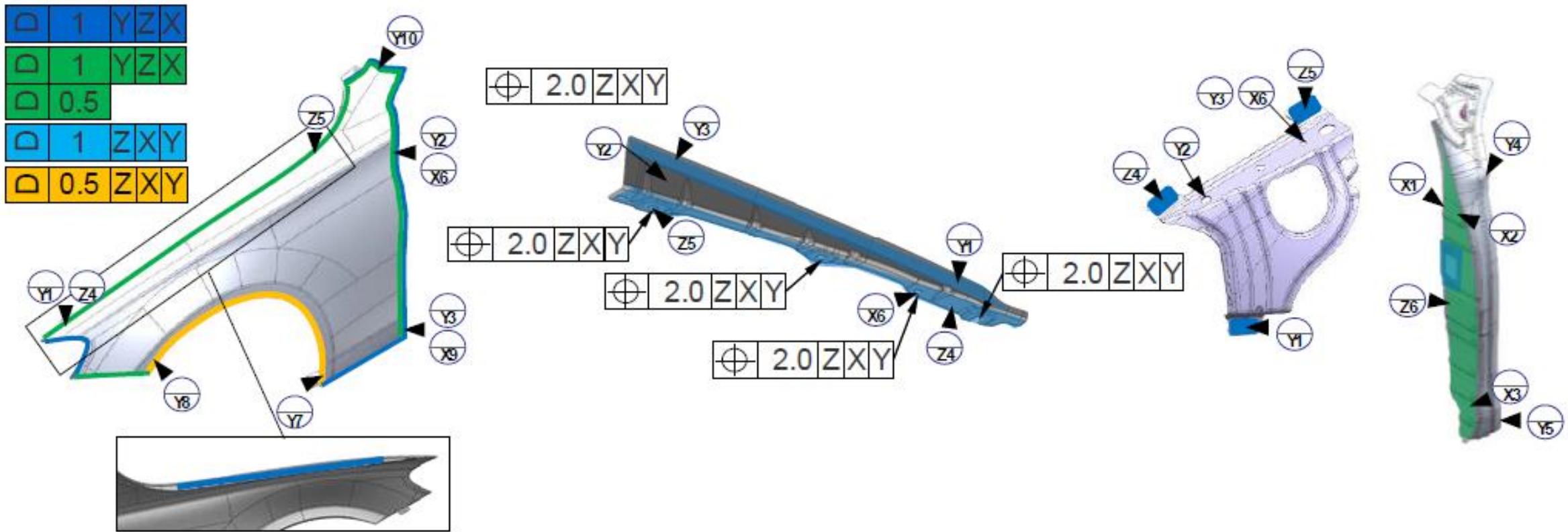
Institut für Technische  
Produktentwicklung



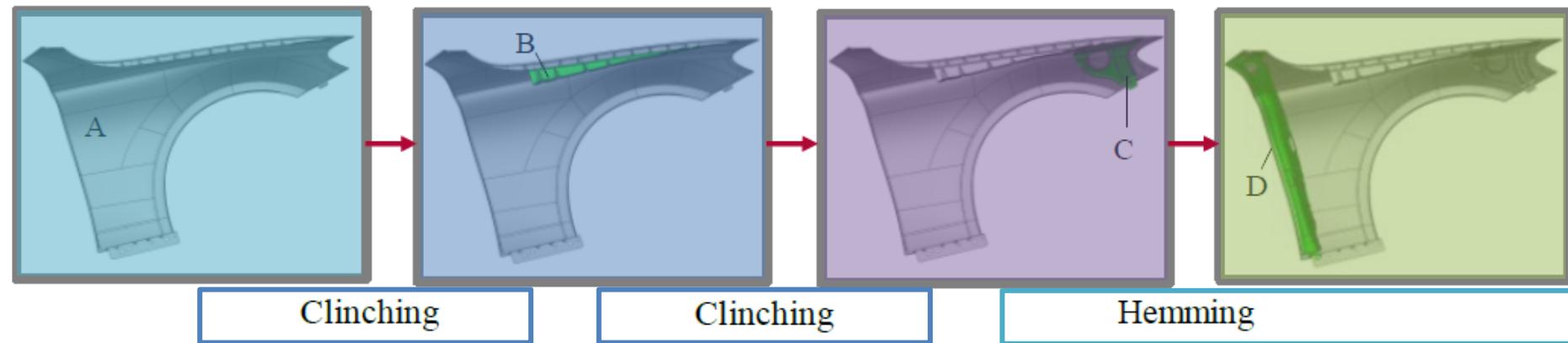
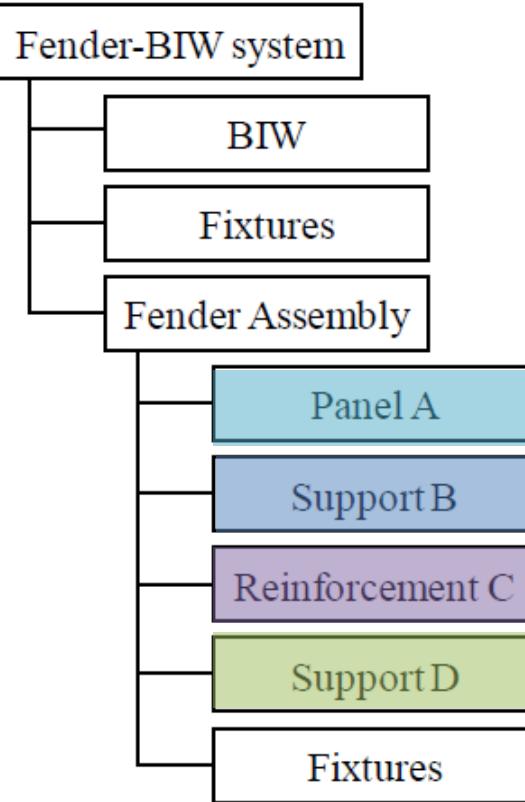
# Fender - Simulate Two Production Scenarios



# Fender part Tolerances



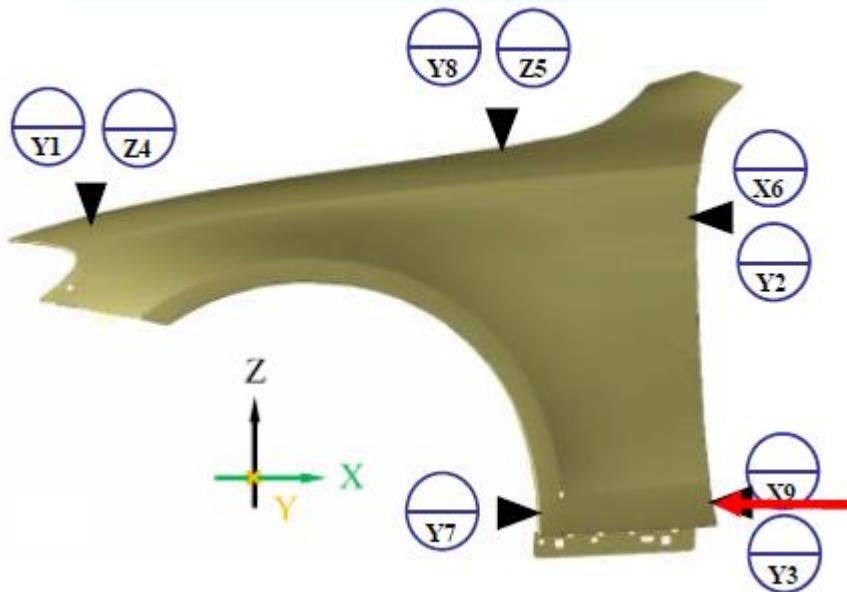
# Fender Assembly – Modeling



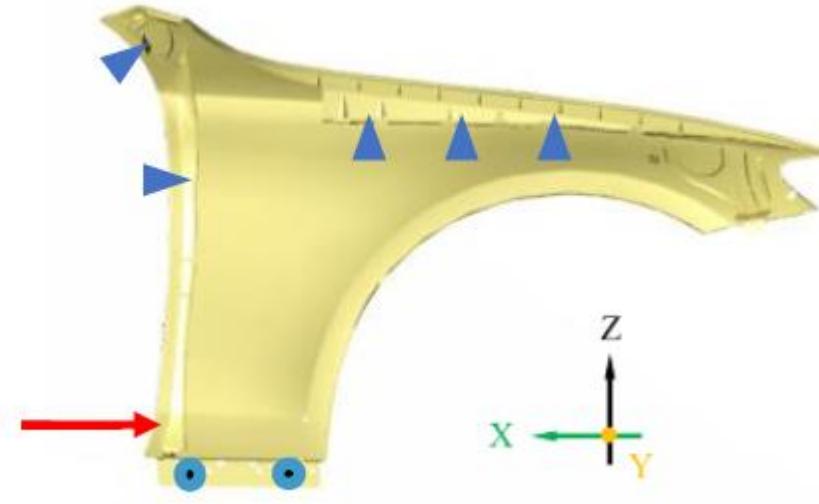
Manufacturing process	Compliant moves	Actions in the simulation model
Clinching	Join	Combine the stiffness
Fixture close/release	Clamp	Fix the degree of freedom (DOF)
Hemming	Join	Combine the stiffness

# Scenario Study

Scenario 1: Fender – Fixtures



Scenario 2: Fender – BIW



► : Fixtures

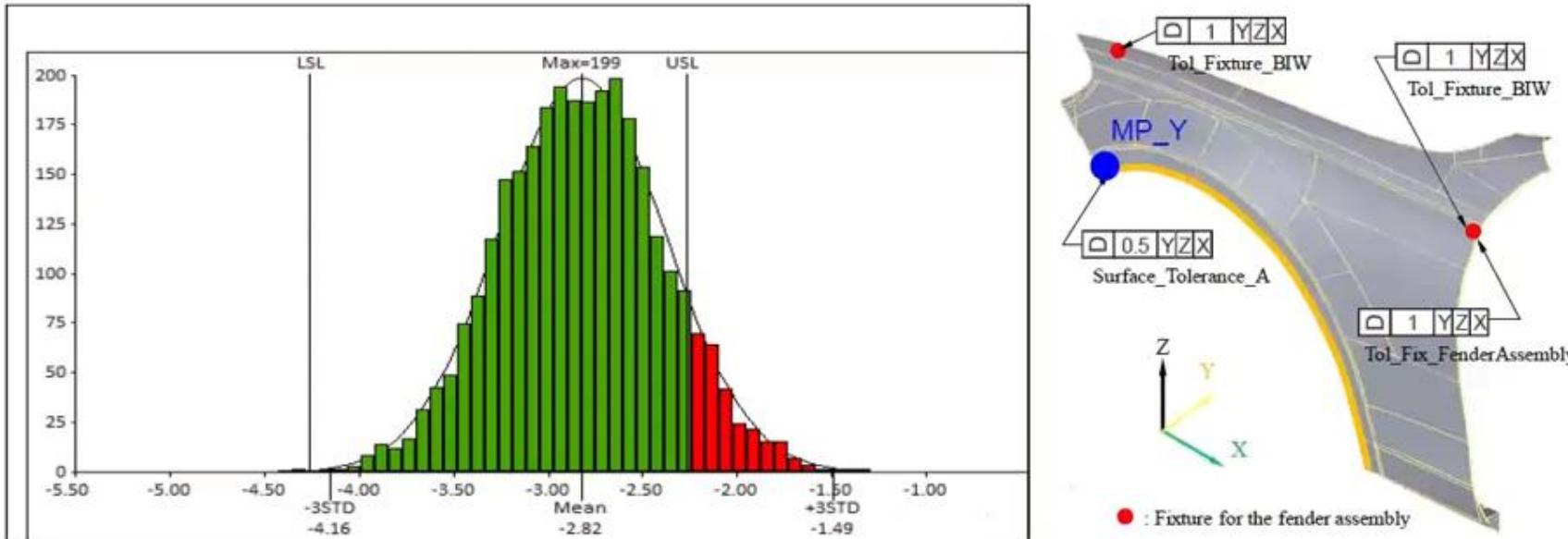
(Y) (X) (Z) : Reference directions

► : Bolts

● : Bolts in +Y direction

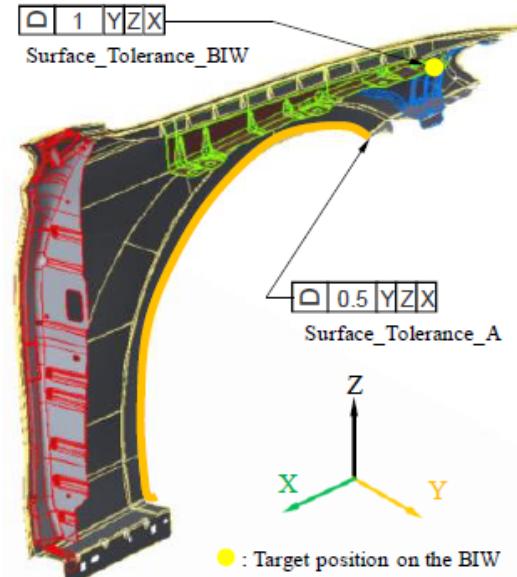
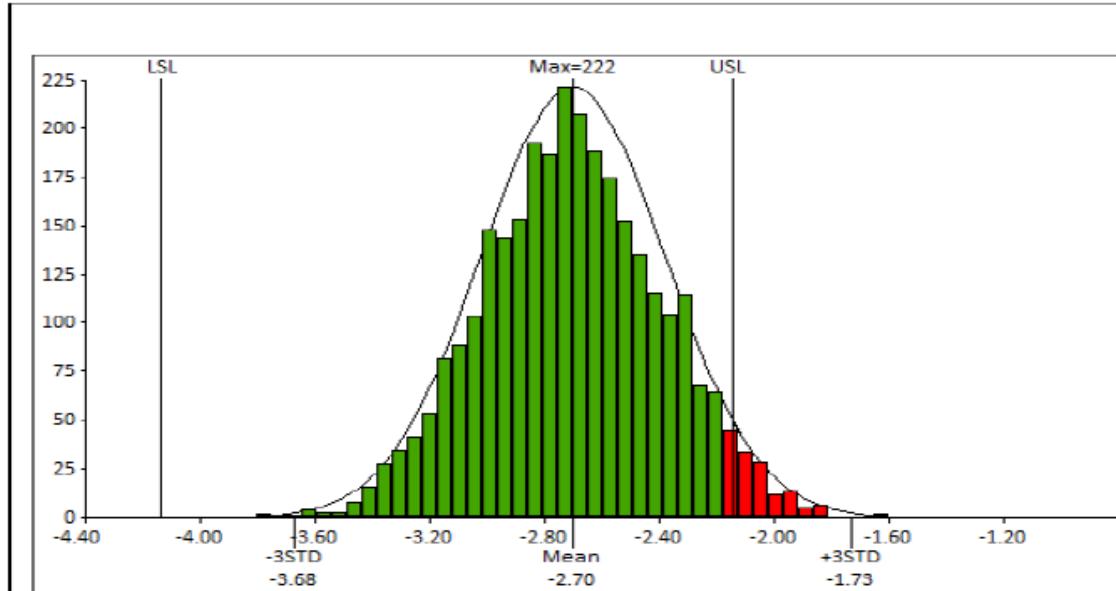
2 mm deviation in (-X) direction  
is constantly observed at the bottom of the fender (the red arrow)

# Scenario 1: Fixture



MP_Y	Index	Contributor	Feature	Part	Range	Contribution
Runs = 3000	1	Tol_Fixture_BIW	X22_AL_X6_BIW	1 BIW_Fix	M:1.000	58.18%
Nominal = -3.26	2	Tol_Fix_FenderAssembly	X22_AL_X6_Fen_10	ZB_Fender_Fix	M:1.000	11.06%
Mean = -2.82	3	Surface_Tolerance_A	Y14_AS_Y8_Fen_10	Fender_Panel_A	M:0.500	10.59%
STD = 0.44	4	Tol_Fixture_BIW	Z10_AL_Z4_BIW	ZB_Fender_Fix	M:1.000	10.43%
6STD = 2.67	5	Tol_Fixture_BIW	Y11_AL_Y1_BIW	ZB_Fender_Fix	M:1.000	5.39%
LSL = -4.26	6	Tol_Fix_FenderAssembly	Y11_AL_Y1_Fen_10	ZB_Fender_Fix	M:1.000	0.61%
USL = -2.26	7	Tol_Fixture_BIW	Y26_AS_Fen_10_Y8_ZB	BIW_Fix	M:1.000	0.56%
Est.Type = Normal	8	Tol_Fixture_BIW	Z27_AL_Z5_BiW	BIW_Fix	M:1.000	0.51%
Est.Low = -4.16	9	Tol_Fix_FenderAssembly	X19_AL_X9_Fen_10	ZB_Fender_Fix	M:1.000	0.32%
Est.High = -1.49	10	Tol_Fix_FenderAssembly	Z10_AL_Z4_Fen_10	ZB_Fender_Fix	M:1.000	0.30%
Est.Range = 2.67	Sum of Remaining 50 Contributors = 2.02%					

# Scenario 2: Bolting BIW

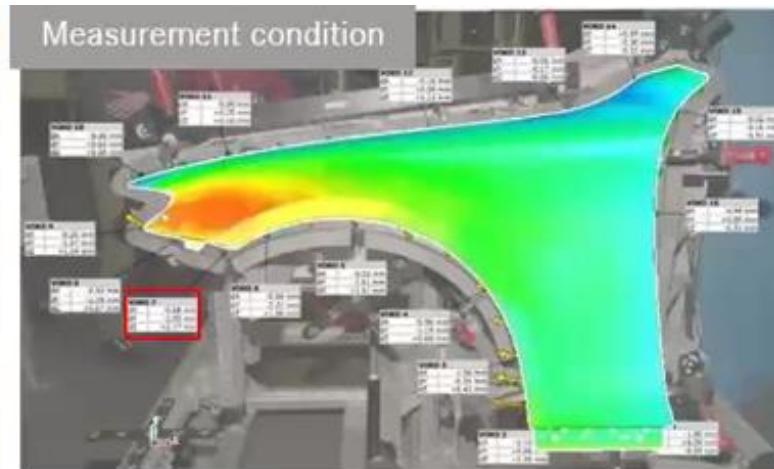


MP_Y	Index	Contributor	Feature	Part	Range	Contribution
Runs = 3000	1	Surface_tolerance_A	Y14_AS_Y8_Fen_10	Fender_Panel_A	M:0.500	19.14%
Nominal = -3.14	2	Surface_tolerance_BIW	Tol5_RF_10	BIW_Body	M:1.000	13.78%
Mean = -2.70	3	Surface_tolerance_CP	Join13_Fend_to_RF	Fender_Panel_A	M:1.000	8.53%
STD = 0.33	4	Surface_tolerance_CP	Join13_Fend_to_RF	Fender_Support_B	M:1.000	8.53%
6STD = 1.95	5	Tol_Fix_FenderAssembly	Z10_AL_Z4_Fen_10	ZB_Fender_Fix	M:1.000	7.29%
LSL = -4.14	6	Tol_Fix_FenderAssembly	Y14_AS_Y8_Fen_10	ZB_Fender_Fix	M:1.000	6.89%
USL = -2.14	7	Position_tolerance_B	Tol5_RF_10	Fender_Support_B	M:1.000	6.12%
Est.Type = Normal	8	Tol_Fix_FenderAssembly	X22_AL_X6_Fen_10	ZB_Fender_Fix	M:1.000	5.06%
Est.Low = -3.68	9	Tol_Fix_FenderAssembly	Y11_AL_Y1_Fen_10	ZB_Fender_Fix	M:1.000	4.13%
Est.High = -1.73	10	Surface tolerance D	Join20_RF_to_SC	Fender_Support_D	M:1.000	2.31%
Est.Range = 1.95	Sum of Remaining 49 Contributors = 18.23%					

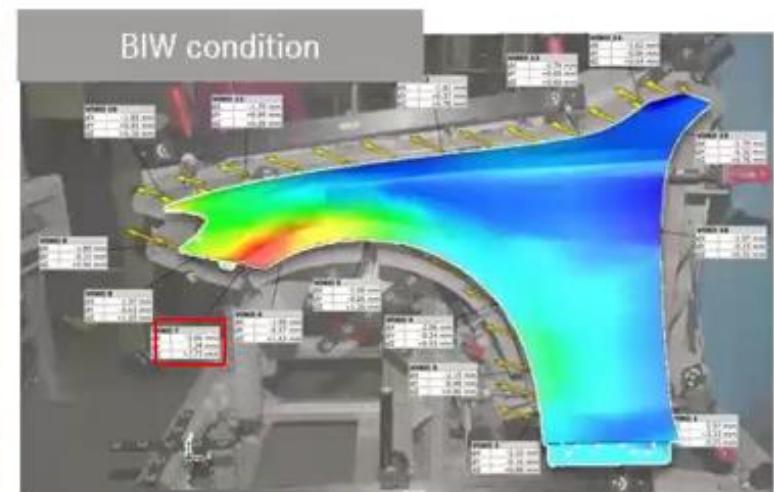
# Simulation Validation



Results for MP in Scenario 1	Measurement			Simulation		
	X	Y	Z	X	Y	Z
Displacement at X9 [mm]	-1.80	+0.09	-0.59	-2.00	0.00	0.00
Nominal	0.00	0.00	0.00	-0.63	<b>-3.26</b>	+2.12
Deformation [mm]	-0.68	<b>-2.93</b>	+2.17			
Mean value [mm]				-0.61	<b>-2.82</b>	+2.13
Deviation range [mm]				0.52	<b>2.67</b>	1.97

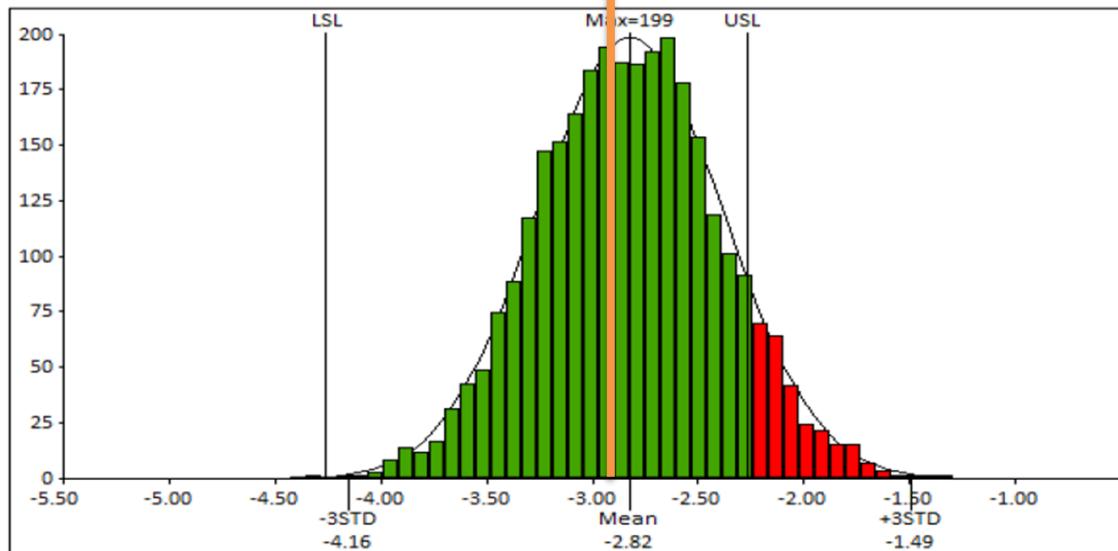


Results for MP in Scenario 2	Measurement			Simulation		
	X	Y	Z	X	Y	Z
Displacement at X9 [mm]	-1.97	+0.11	-0.25	-2.00	0.00	0.00
Nominal	0.00	0.00	0.00	-0.58	<b>-3.14</b>	+2.45
Deformation [mm]	-0.64	<b>-3.24</b>	+2.72			
Mean value [mm]				-0.56	<b>-2.70</b>	+2.46
Deviation range [mm]				0.28	<b>1.95</b>	1.18



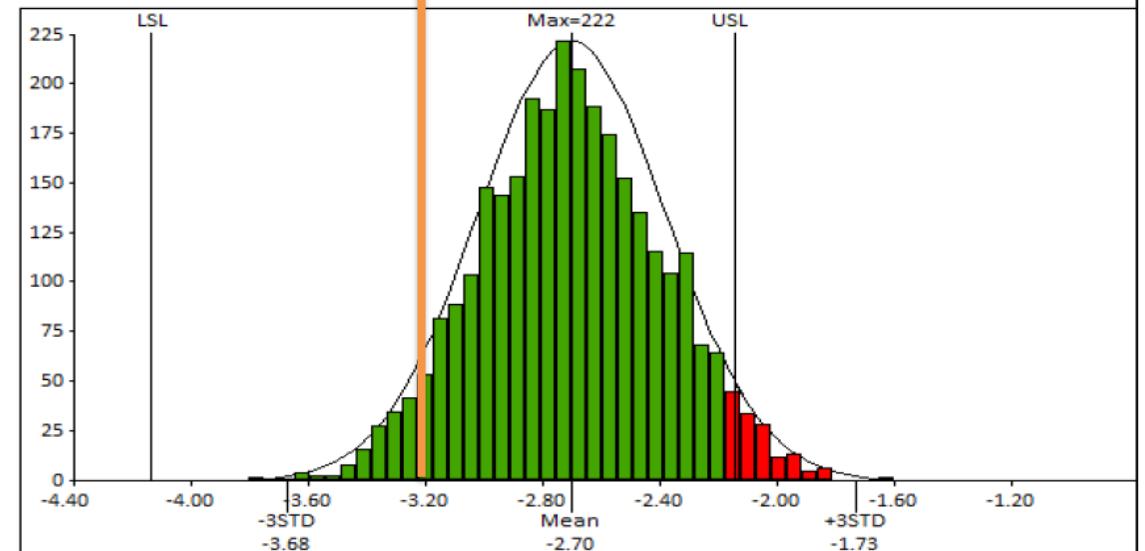
# Simulation & Measurement

Measured Y Deformation



Scenario 1

Measured Y Deformation



Scenario 2

## Conclusions:

- An approach is developed to implement the Elastic Tolerance Simulation
- Proposed modeling approach is applied in industrial scenarios & Validated by real measurement data

## Outlook

- Automatic integration process of the CAD points to Simulation System
- A Simulation System Containing more compliant parts
- Non Linear effects of the clinching/hemming process

# Questions?